Amendment to the Claims:

The listing of claims will replace all prior versions and listings of claims in the application:

Claims 1-24. (canceled).

Claim 25. (currently amended): ~~A cone pulley transmission according to claim 23,~~

An infinitely variable cone pulley transmission for generating of axial contact pressure forces upon a traction means, comprising:

an input side shaft and an output side shaft;

cone pulleys arranged on the input side and output side shafts, the traction means rotating between the cone pulleys;

means arranged on the transmission shafts, for exerting forces in axial direction upon respectively one cone pulley that can be displaced axially along the respective shaft, the exerting means including hydraulic tensioning means provided on input side shaft for adjusting and maintaining the transmission ratio and a spring supported tensioning means that is braced against a support, fixed relative to the output side shaft provided on the output side shaft,

wherein the cone pulleys on the output side shaft include an axially fixed cone pulley and an axially movable cone pulley with an extended hub are jointly arranged on the output side shaft,

wherein the cone pulleys on the input side and output side shafts are rotationally connected and jointly rotate on the respective shaft and are coupled to said the respective shaft and the exerting means exerts a contact pressure that depends on the rotational moment or the rotational moment and the transmission ratio,

wherein the exerting means of the output side shafts includes a cam sleeve that is fixedly connected to the output side shaft, a cam sleeve formed by the free end of an extended hub and roll bodies for transmitting force,

wherein said roll bodies are inserted between opposite arranged cam curves and rotate around roll body axes extending in radial direction, relative to the output side shaft,

wherein the exerting means further includes rings for guiding and holding the roll bodies at a mutual distance to each other in the axial center region between the cam curves with the aid of a spring that is arranged coaxial on the extended hub,

wherein the roll bodies include pinions that are coaxial to a rotational axes of the pinions and project in radial direction relative to the output side shaft from the roll bodies for engaging corresponding recesses, and

wherein, in radial direction relative to the output side shaft, wherein the rings include a guide ring and a holding ring, the guide ring being in the form of a hollow-cylindrical sleeve is arranged outside of the roll bodies, wherein the guide ring encloses the roll bodies and the extended hub, and the guide ring can be displaced in axial direction along the extended hub, but is positioned such that the guide ring cannot rotate relative to the extended hub, wherein at least some of the pinions on the roll bodies are outward projecting and are positioned to rotate in circumferential slots of the guide ring that extend along a radial plane of the output side shaft, the slots having an axial width corresponding to the diameter of the pinions and are held in the region of the axial center between the cam sleeves, wherein, in circumferential direction, the length of the circumferential slots corresponds to at least half the maximum mutual circumferential path for the cam sleeves of the exerting means and wherein the guide ring has an end facing the axially movable cone pulley forming an extension onto the spring, such that moving in the same direction, the guide ring respectively traverses essentially half the axial distance traversed by the movable cone pulley.

Claim 26. (previously presented): A cone pulley transmission according to claim 25, wherein the spring is supported on one side on the axially movable cone pulley and on the other side on the cam sleeve that is fixedly connected to the output side shaft.

Claim 27. (previously presented): A cone pulley transmission according to claim 25, wherein the spring is a disk spring assembly, including one half which is arranged on the hub and the other half on the guide ring, wherein the spring is supported on the cam sleeve via a hollow-

cylindrical intermediate segment that encircles the guide ring and wherein the guide ring has a radially outward pointing collar on one end that is captured between two halves of the disk spring assembly.

Claim 28. (previously presented): A cone pulley transmission according to claim 25, wherein the guide ring is provided with at least one groove that extends parallel to the axis of the output side shaft, wherein the hub includes a radial pin engaging the at least one groove to prevent rotation.

Claim 29. (currently amended): ~~A cone pulley transmission according to claim 23,~~

An infinitely variable cone pulley transmission for generating of axial contact pressure forces upon a traction means, comprising:

an input side shaft and an output side shaft;

cone pulleys arranged on the input side and output side shafts, the traction means rotating between the cone pulleys;

means arranged on the transmission shafts, for exerting forces in axial direction upon respectively one cone pulley that can be displaced axially along the respective shaft, the exerting means including hydraulic tensioning means provided on input side shaft for adjusting and maintaining the transmission ratio and a spring supported tensioning means that is braced against a support, fixed relative to the output side shaft provided on the output side shaft,

wherein the cone pulleys on the output side shaft include an axially fixed cone pulley and an axially movable cone pulley with an extended hub are jointly arranged on the output side shaft,

wherein the cone pulleys on the input side and output side shafts are rotationally connected and jointly rotate on the respective shaft and are coupled to said the respective shaft and the exerting means exerts a contact pressure that depends on the rotational moment or the rotational moment and the transmission ratio,

wherein the exerting means of the output side shafts includes a cam sleeve that is fixedly

connected to the output side shaft, a cam sleeve formed by the free end of an extended hub and roll bodies for transmitting force,

wherein said roll bodies are inserted between opposite arranged cam curves and rotate around roll body axes extending in radial direction, relative to the output side shaft,

wherein the exerting means further includes rings for guiding and holding the roll bodies at a mutual distance to each other in the axial center region between the cam curves with the aid of a spring that is arranged coaxial on the extended hub,

wherein the roll bodies include pinions that are coaxial to a rotational axes of the pinions and project in radial direction relative to the output side shaft from the roll bodies for engaging corresponding recesses, and

wherein, outside of the roll bodies and in radial direction relative to the output side shaft, wherein the rings include a guide ring that encompasses the roll bodies is arranged in the form of at least one assembly of axially side-by-side arranged, ring-shaped corrugated springs, having reciprocal undulations in axial direction along the circumference, wherein the guide ring is captured in axial direction between a rotating collar supported by the hub and a collar supported by the cam sleeve that is fixedly connected to the shaft and is kept axially centered relative to the exerting means and wherein at least some of the pinions on the roll bodies are radially outward projecting pinions positioned rotating in the axial center of said guide ring.

Claim 30. (previously presented): A cone pulley transmission according to claim 29, wherein the guide ring includes two identical corrugated spring assemblies, arranged axially side-by-side, wherein the corrugated springs of each assembly are braced against each other by undulation peaks that face each other and are fixedly connected, and wherein the pinions on the roll bodies are positioned so as to rotate between the corrugated spring assemblies.

Claim 31. (previously presented): A cone pulley transmission according to claim 30, wherein the rings include a holding ring arranged outside of the roll bodies, between the roll bodies

and the guide ring of the corrugated springs, and wherein the holding ring is provided with a rotating collar that projects at the center radially outward from the holding ring and engages between the corrugated spring assemblies.

Claim 32. (previously presented): A cone pulley transmission according to claim 31, wherein the bores in the holding ring are designed to accommodate the pinions on the roll bodies and extend through the collar.

Claim 33. (previously presented): A one pulley transmission according to claim 31, wherein the axial width of the collar corresponds to the width of the roll body pinions.

Claim 34. (previously presented): A cone pulley transmission according to claim 32, wherein an axial width of the collar corresponds to a width of the roll body pinions.

Claim 35. (previously presented): A cone pulley transmission according to claim 30, wherein the spring is arranged on the hub and is braced against the axially movable cone pulley and the cam sleeve that is fixedly connected to the shaft with the aid of an essentially hollow-cylindrical intermediate segment that encompasses the guide ring.

Claim 36. (previously presented): A cone pulley transmission according to claim 35, wherein the collar supported by the cam sleeve fixed relative to the shaft and the intermediate segment are combined to form one component.

Claims 37 and 38. (canceled).